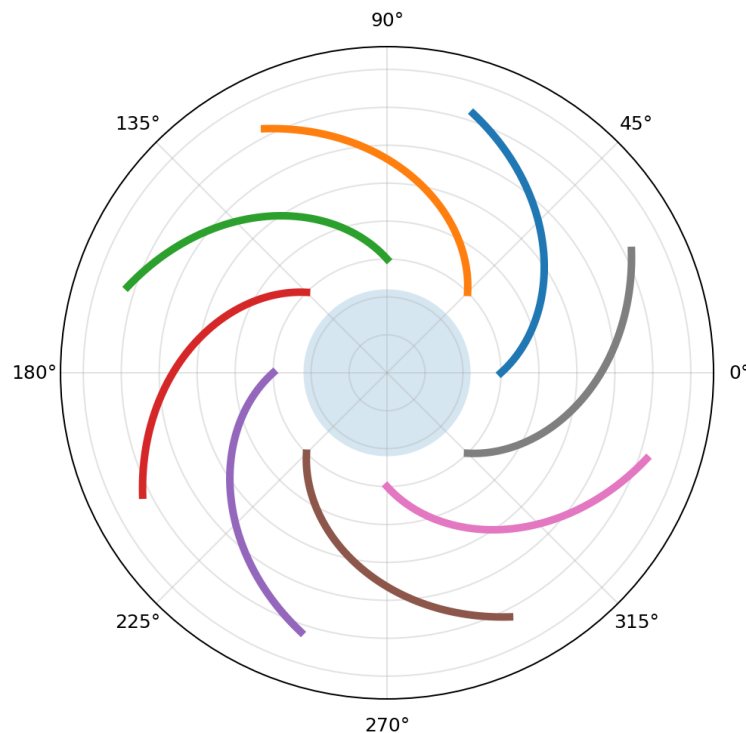


Application of Polar Coordinate Graphs for Industrial Design

Chung S Lee¹, Brendon M Son²

Conceptual Radial Impeller Constructed from Repeated Polar Curves



Cover Figure. Conceptual radial impeller generated from repeated polar-coordinate curves.

Abstract

Polar coordinate graphs provide a compact mathematical language for products whose geometry is organized around a center, axis, rotation, or angular sequence. This paper examines how equations of the form $r = f(\theta)$ can support industrial design from early form exploration through parametric CAD, engineering optimization, and manufacturing. Applications include impellers, turbine and fan components, radial vents and grilles, cams, spiral housings, rotary interfaces, decorative perforations, and other rotationally organized products. The study develops a design methodology based on circles, rose curves, cardioid/limaçon families, Archimedean spirals, logarithmic spirals, and Fourier-type radial profiles. It also explains conversion to Cartesian coordinates, geometric constraints, curvature considerations, repeated angular modules, and the extension of two-dimensional polar curves into three-dimensional surfaces. Published CAD and turbomachinery literature demonstrates the practical relevance of coordinate-based parameterization, NURBS/B-spline surfaces, circular-root relations, design automation, reverse engineering, and five-axis manufacturing. The paper concludes that polar graphs are most valuable not as isolated mathematical illustrations but as controllable design generators that connect aesthetics, function, computation, and fabrication.

Keywords: polar coordinates; industrial design; parametric design; CAD/CAM; spiral geometry; radial symmetry; impeller; turbine blade; cam; computational design

1. Introduction

Industrial products frequently contain geometry that is naturally described by distance from a center and angle about that center. Fans, rotors, knobs, pumps, turbines, loudspeaker grilles, watch components, circular interfaces, and rotary mechanisms are obvious examples. Even when the final object is three-dimensional, its planform, section, sweep, pitch, perforation field, or repeated feature arrangement may be governed by angular position. Cartesian coordinates can represent such geometry, but polar coordinates often expose the design logic more directly.

In a polar system a point is represented by (r, θ) , where r is radial distance and θ is angular position. The basic conversion is $x = r \cos \theta$ and $y = r \sin \theta$. This representation allows a designer to treat radius as a controllable function of angle, $r = f(\theta)$. The equation therefore becomes a parametric design rule: changing a coefficient changes the form globally and predictably.

Current CAD research reinforces the broader value of parameterized geometry. Recent turbomachinery work describes blade parameterization using geometric variables, B-splines and NURBS, with integration into optimization workflows, while earlier wind-turbine CAD research explicitly uses the polar circle relations $x = r \cos(\theta)$, $y = r \sin(\theta)$ for a circular blade root. [1,2]

2. Mathematical Foundation

2.1 Polar-to-Cartesian Representation

For a polar design curve $r = f(\theta)$, the corresponding planar parametric curve is

$$x(\theta) = f(\theta) \cos \theta, \quad y(\theta) = f(\theta) \sin \theta$$

This conversion is important because most CAD kernels ultimately store curves as Cartesian points, splines, or NURBS. A polar equation can therefore serve as the high-level design generator while the resulting sampled or exact geometry is transferred into conventional CAD representations.

2.2 Useful Curve Families

Polar family	Typical equation	Industrial-design use
Circle	$r = R$	Hubs, rings, bearings, circular roots, dial geometry
Rose curve	$r = a \cos(k\theta)$	Vents, grilles, repeated petals, radial cutouts
Cardioid / limaçon	$r = a + b \cos \theta$	Eccentric housings, handles, acoustic openings
Archimedean spiral	$r = a + b\theta$	Coils, scroll paths, grooves, progressive spacing
Logarithmic spiral	$r = ae^{(b\theta)}$	Flow paths, radial blades, self-similar sweep
Fourier radial profile	$r = a_0 + \sum (a_n \cos n\theta + b_n \sin n\theta)$	Cams, ergonomic rotary contours, optimized boundaries

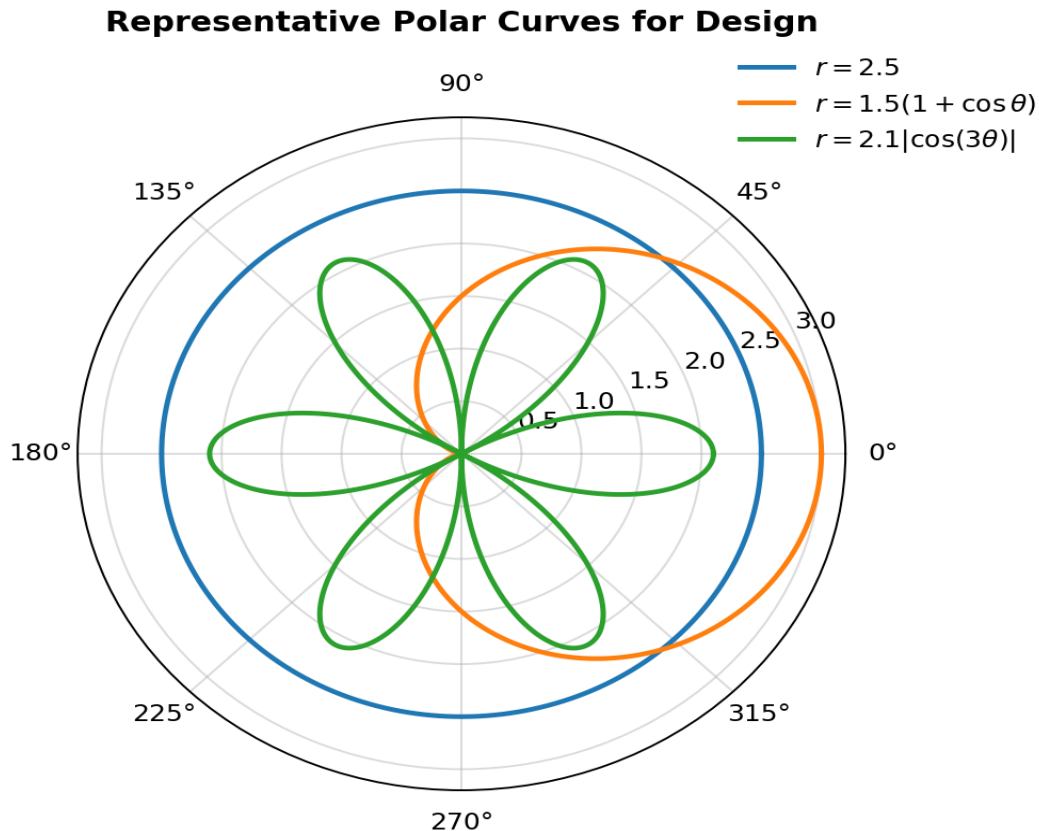


Figure 1. Representative polar curve families used as design generators.

3. Why Polar Graphs Are Valuable in Industrial Design

Polar graphs align the mathematics with the designer's intent whenever angular repetition, radial growth, sweep, eccentricity, or rotational motion is central to the product. Four advantages are especially important. First, symmetry is encoded efficiently: a k-petal rose curve can generate repeated features without drawing each feature independently. Second, proportions become parameter-driven; radius, pitch, phase, lobe count, and spiral growth can be exposed as design variables. Third, polar equations support continuous variation rather than only discrete sketch constraints. Fourth, the same variables can be passed into optimization, simulation, and manufacturing scripts.

POLAR-COORDINATE INDUSTRIAL DESIGN WORKFLOW

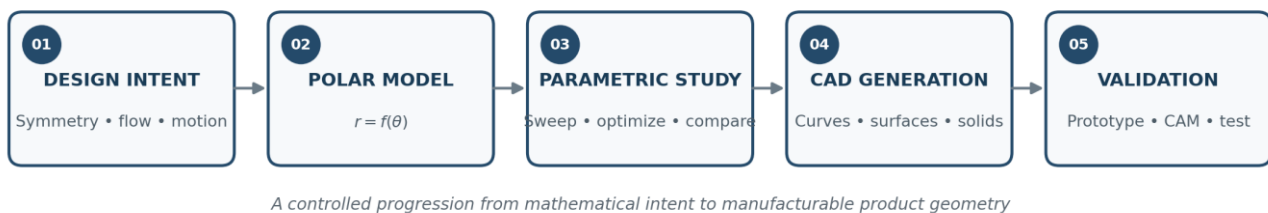


Figure 2. Proposed workflow linking mathematical form generation to CAD/CAM validation.

4. Application I: Radial Impellers, Fans, and Turbomachinery

Rotating flow machinery is one of the clearest engineering contexts for polar thinking. Blade locations are naturally indexed by angle, hub and shroud boundaries are radial, and blade sweep can be described by an angular-radius relationship. A simple logarithmic spiral, $r = ae^{(b\theta)}$, produces a constant-angle spiral and can serve as a conceptual camber-line generator. More advanced industrial blades require airfoil thickness, three-dimensional stacking, twist, structural constraints, and fluid-dynamic optimization, so the polar curve is best understood as one layer in a larger parameterization.

Polar Geometry Applied to a Radial Impeller

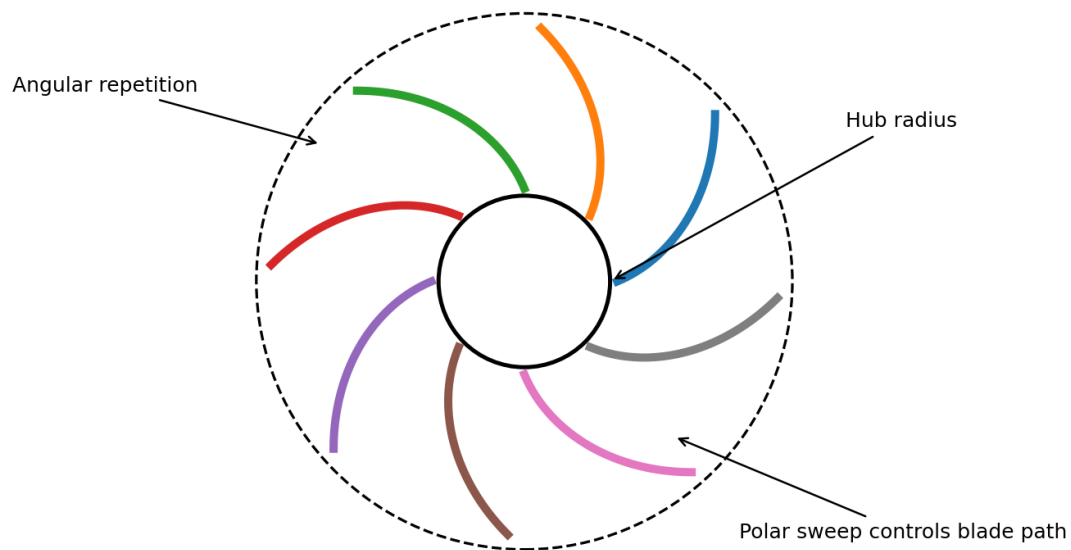


Figure 3A. Industrial-design interpretation of polar sweep, hub radius, and angular repetition in a radial impeller.

Published work on gas-turbine blade parameterization uses geometric variables together with B-spline and NURBS representations to create two-dimensional sections and connect them into three-dimensional blade surfaces. A unified parameterization method has also been demonstrated for axial, radial, and mixed-flow blades and integrated with gradient-based optimization. [1,3] Wind-turbine CAD research similarly reports a circular root generated using the polar circle relations before smooth connection to aerodynamic surfaces. [2]

Spiral Families for Radial Product Geometry

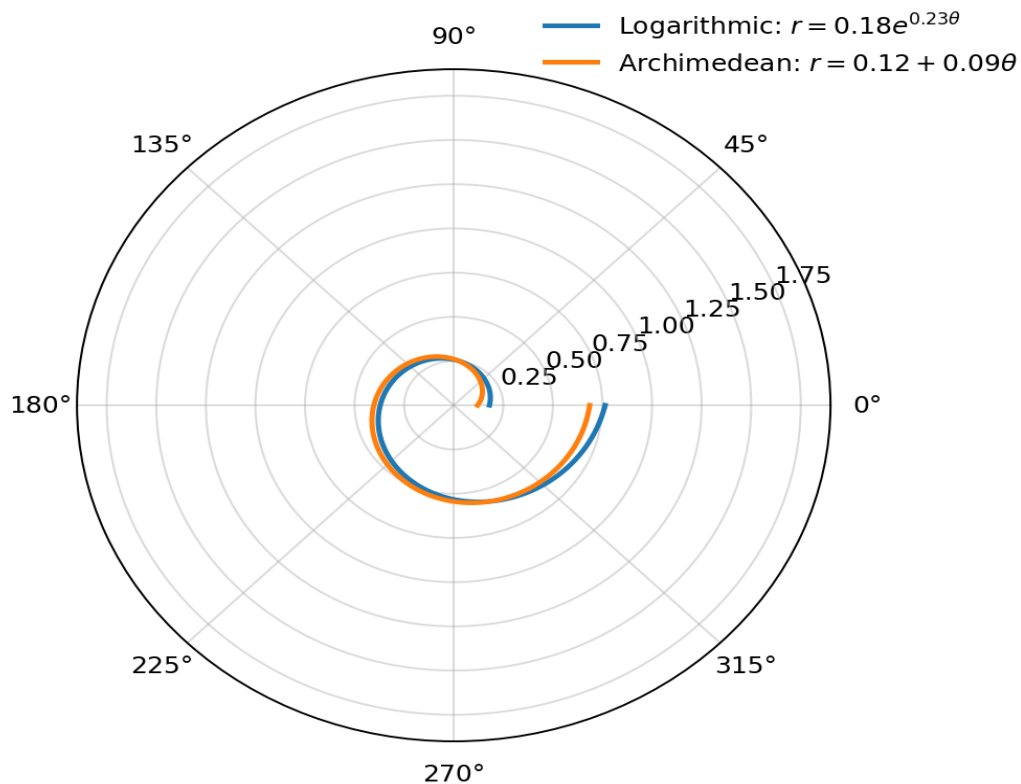


Figure 3. Logarithmic and Archimedean spirals as candidate generators for radial sweep and progressive paths.

Conceptual Radial Impeller Constructed from Repeated Polar Curves

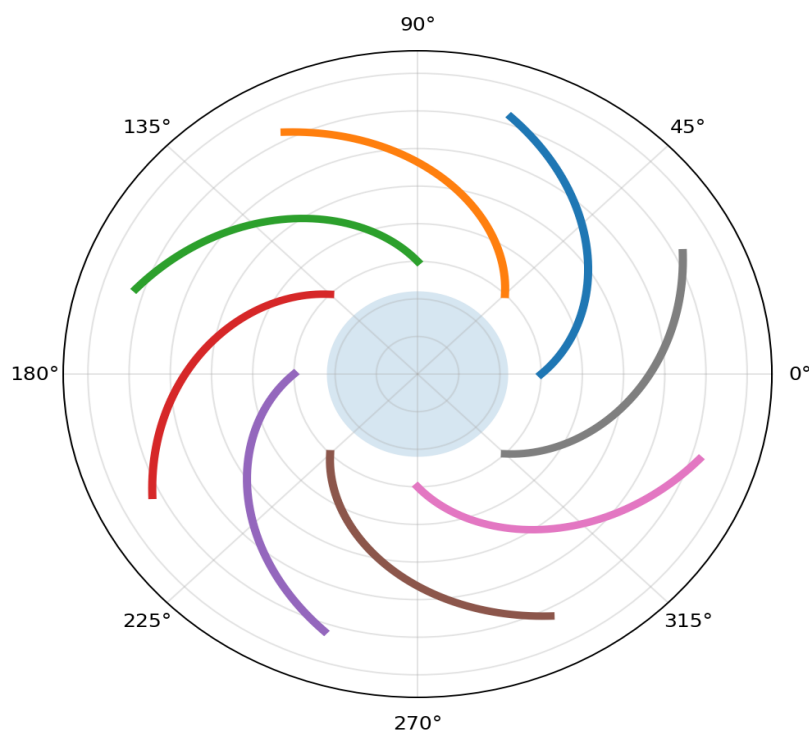


Figure 4. Conceptual eight-blade impeller generated by rotating a base polar curve.

5. Application II: Vents, Grilles, Perforations, and Product Styling

Consumer and architectural products often require radial perforation patterns that must satisfy both visual and functional criteria. A rose curve $r = a \cos(k\theta)$, or its absolute-value form, gives direct control over lobe count and amplitude. Concentric circles can be combined with such curves to establish zones for open area, ribs, fasteners, or acoustic passages. Phase shifts rotate the pattern without redrawing it, while amplitude modulation can create hierarchy between center and perimeter.

Polar Rose Curves for Product Vents and Grilles

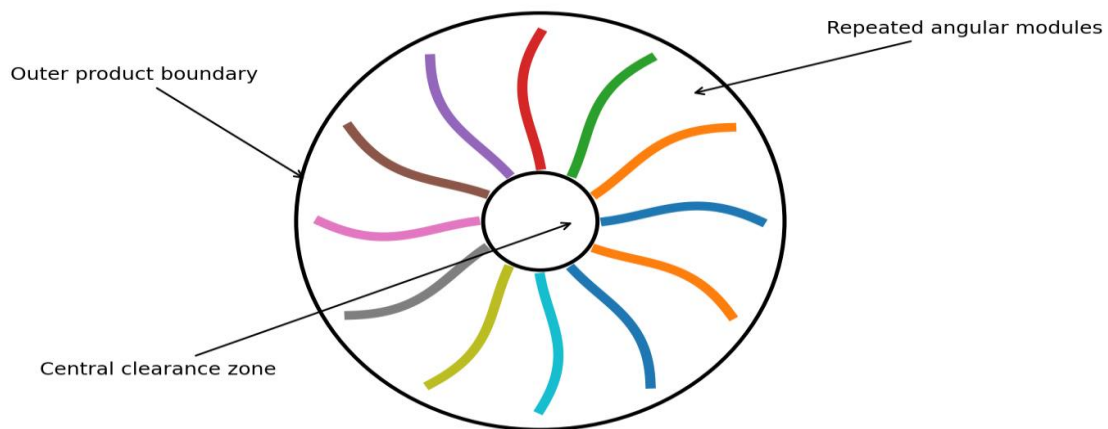


Figure 5A. Product grille concept showing repeated angular modules, central clearance, and an outer circular boundary.

Radial Vent / Grille Concept Using a Rose Curve

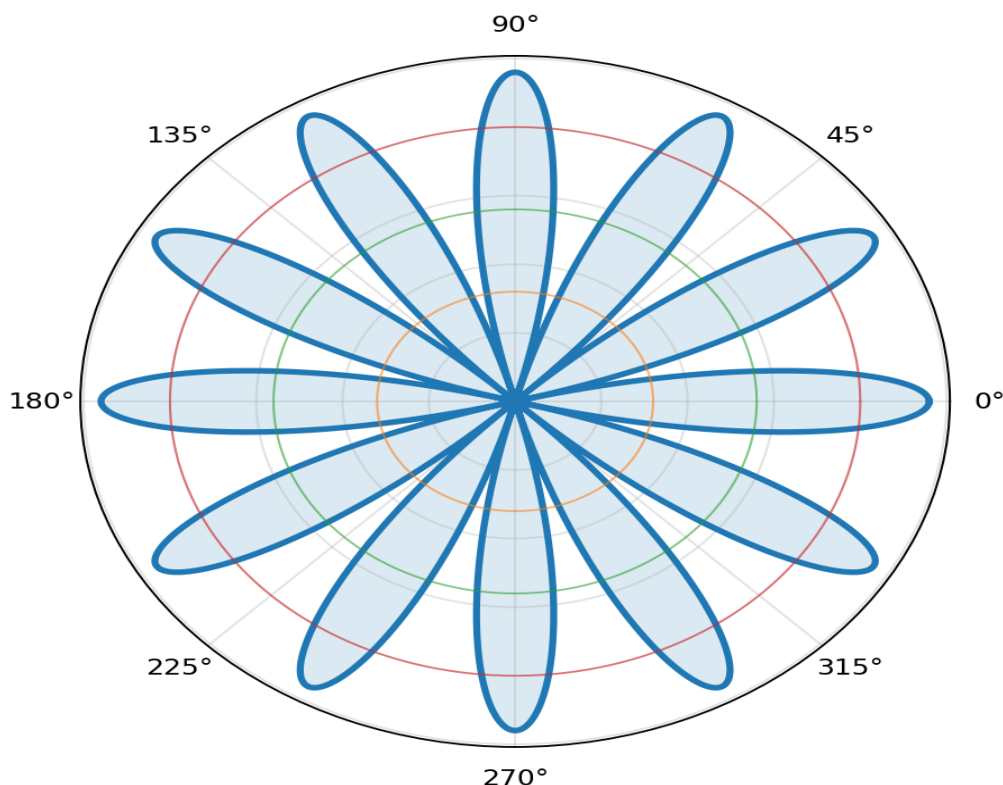


Figure 5. Rose-curve concept for a radial vent or loudspeaker grille.

For manufacturable products, the mathematical curve should be offset into finite-width ribs or cutouts and then checked for minimum wall thickness, tool diameter, draft, molding flow, stress concentration, and open-area requirements. Thus, the polar equation generates design intent, while engineering constraints regularize the final geometry.

6. Application III: Cams and Rotary Motion Profiles

A cam is naturally interpreted as a radial distance that changes with shaft angle. A general Fourier-style expression, $r(\theta) = a_0 + \sum (a_n \cos n\theta + b_n \sin n\theta)$, can represent smooth periodic profiles with adjustable harmonics. Industrial design benefits from this representation because motion character, dwell regions, eccentricity, and visual profile can be tuned with a small set of coefficients.

Polar Cam Profile and Rotary Motion

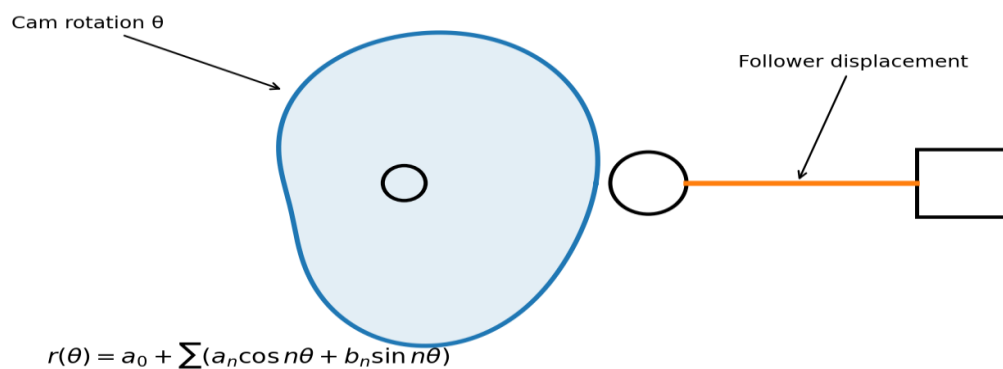


Figure 6A. Cam-and-follower schematic linking a periodic polar radius to rotary motion.

Illustrative Polar Cam Profile

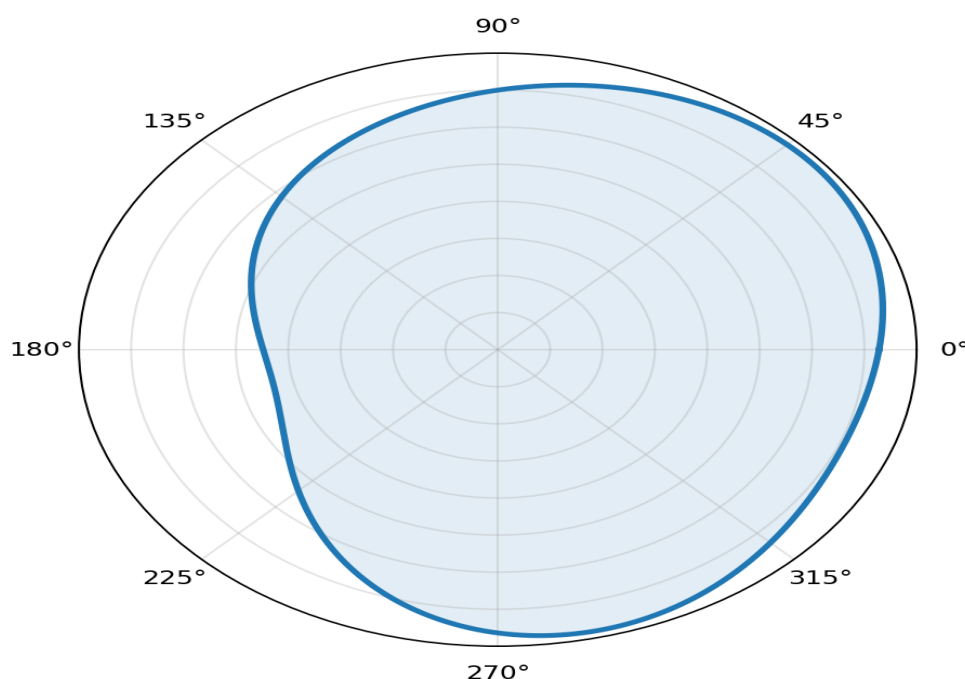


Figure 6. Illustrative smooth cam profile represented as radius versus angular position.

However, a visually smooth polar graph is not automatically a dynamically acceptable cam. Designers must evaluate curvature, pressure angle, follower acceleration, jerk, contact stress, lubrication, and manufacturability. Polar mathematics provides the geometric starting point; mechanical design criteria determine whether the profile is usable.

7. Application IV: Spiral Housings, Grooves, Coils, and Routing

Archimedean spirals are useful when approximately constant radial spacing per revolution is desired, because r increases linearly with θ . This makes them suitable as conceptual generators for grooves, planar coils, spiral tracks, decorative channels, and progressive routing. Logarithmic spirals instead scale multiplicatively with angle and are useful where self-similar growth or constant spiral angle is desired.

Spiral Geometry for Housings, Grooves, and Routing

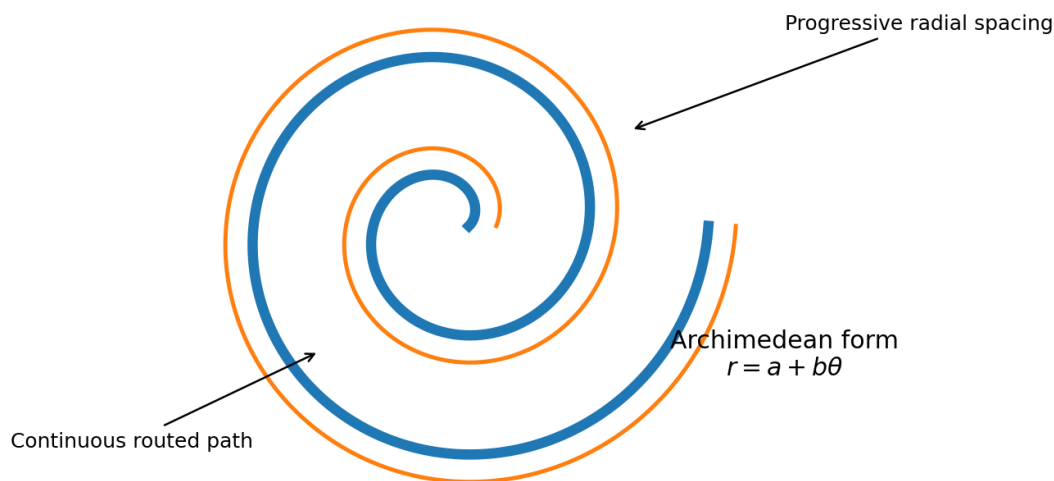


Figure 7. Archimedean spiral concept for progressive grooves, housings, and routed paths.

Research on spiral-type free-form CAD/CAM models has connected parametric spiral geometry with automated three-dimensional modeling and NC program generation, illustrating how mathematically controlled curves can move directly toward manufacturing workflows. [4]

8. From 2-D Polar Curves to 3-D Product Geometry

A two-dimensional polar curve can become a three-dimensional industrial form through several standard operations: extrusion, revolution, lofting between polar sections, sweep along an axial path, radial stacking, and height mapping. For example, a family $r = f(\theta, z)$ allows the radius to vary simultaneously with angle and height. A simple formulation is $r(\theta, z) = r_0(\theta)[1 + \lambda z]$, while more complex models can vary phase, lobe count, or amplitude along z .

From a 2-D Polar Profile to a 3-D Product Surface

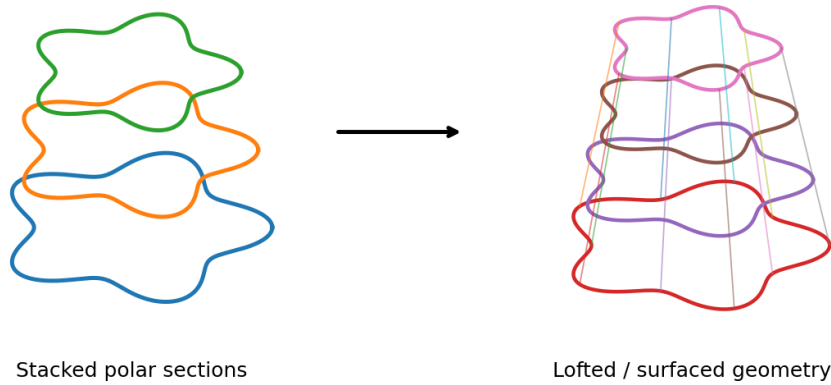


Figure 8. Conceptual transition from stacked two-dimensional polar sections to a three-dimensional lofted product surface.

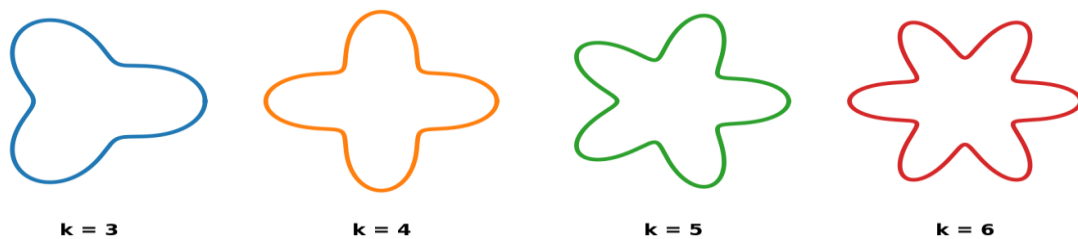
Once section curves are generated, spline or NURBS representations are commonly used to obtain smooth CAD surfaces. Modern turbomachinery parameterization research specifically emphasizes NURBS curve networks and multidisciplinary optimization, while wind-turbine blade modeling demonstrates the importance of smooth G1 connections between structural and aerodynamic regions. [1,2]

9. Parametric CAD and Design Automation

The principal computational advantage of polar graphs is parameter exposure. Instead of manually editing many sketch entities, a designer can define a compact variable set such as $\{R, a, b, k, \phi, t, N\}$, representing base radius, amplitude, growth rate, lobe count, phase, thickness, and repetition count. A script or parametric CAD system can regenerate the product whenever these values change.

Parametric CAD: One Equation, Multiple Design Variants

$$r(\theta) = R + A \cos(k\theta + \phi)$$



Changing one parameter regenerates the entire product family.

Figure 9. Parametric design variants produced by changing the angular-frequency parameter k .

Industrial CAD automation studies in turbine-engine design report that scripting and knowledge-based engineering can reduce design time, enable more feasible design iterations, and improve relational modeling. [5] This supports a broader industrial-design strategy: polar equations should be embedded as editable generative rules rather than treated only as static plotted curves.

10. Manufacturing and Reverse Engineering

Polar-generated geometry must ultimately be translated into manufacturing representations. For planar components, sampled polar points can be converted to Cartesian coordinates and fitted with splines for DXF or similar exchange. For complex three-dimensional parts, surfaces may be exported through standard CAD formats and passed to CAM for multi-axis machining.

Polar Design to Manufacturing Data



$$r=f(\theta) \rightarrow (x,y) \rightarrow \text{CAD geometry} \rightarrow \text{NC/CAM} \rightarrow \text{physical validation}$$

Figure 10. Manufacturing pipeline from a polar equation to Cartesian geometry, CAD/CAM data, and a physical part.

Research on centrifugal compressor impellers and turbine blades has demonstrated reverse-engineered control points, CAD surface construction, tool-path planning, NC post-processing, and five-axis machining simulation. [6] Laser-scanned turbine data have likewise been transformed into smooth CAD surfaces and meshes for CFD, demonstrating the importance of preserving as-built geometry while producing usable computational models. [7]

11. Design Constraints and Evaluation Metrics

A polar design should be evaluated against both geometric and industrial criteria:

Engineering Constraints Applied to Polar-Generated Forms

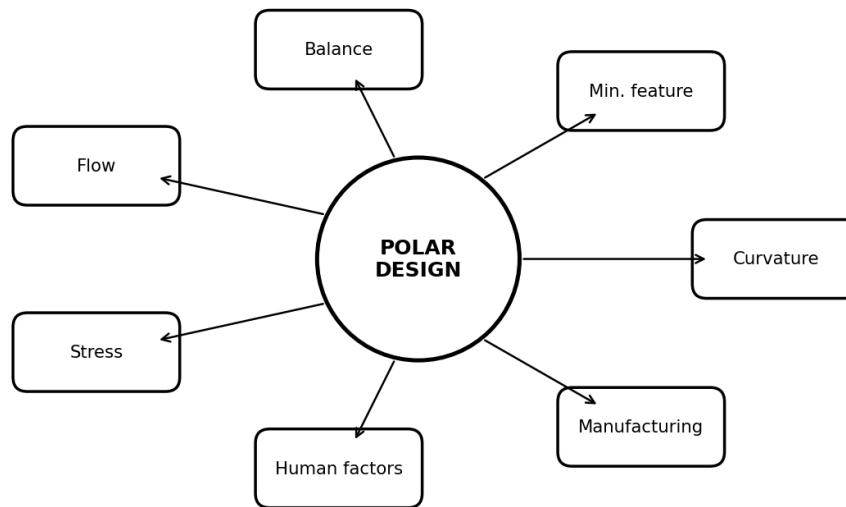


Figure 11. Key engineering constraints surrounding a polar-generated industrial form.

- Continuity: avoid unintended cusps or discontinuities in $r(\theta)$ and its derivatives.
- Curvature: limit excessively small radii of curvature that create stress or machining problems.
- Minimum feature size: maintain manufacturable rib, wall, slot, and edge dimensions.
- Symmetry and balance: ensure rotating products satisfy mass-balance and dynamic requirements.
- Flow performance: for fans, impellers, vents, and ducts, validate with appropriate fluid analysis.
- Structural performance: evaluate stress, fatigue, vibration, and attachment regions.
- Human factors: check grip, reach, touch, visual hierarchy, and perceived quality for user-facing products.
- Process compatibility: account for CNC cutter access, additive overhangs, molding draft, sheet-metal limitations, and tolerances.

12. Proposed Industrial Design Methodology

Step	Stage	Design action
1	Define the rotational design intent	Identify the center/axis, functional zones, required symmetry, motion, flow, or visual rhythm.
2	Select a polar family	Choose circle, rose, limacon, Archimedean spiral, logarithmic spiral, or Fourier radial profile.
3	Expose design parameters	Define radius, amplitude, frequency/lobe count, phase, pitch/growth, and thickness as variables.

4	Generate and inspect curves	Plot $r(\theta)$, check self-intersections, curvature, spacing, and boundary conditions.
5	Convert to CAD geometry	Use $x=r \cos\theta$ and $y=r \sin\theta$; fit splines/NURBS or create equation-driven curves.
6	Construct 3-D form	Extrude, loft, sweep, revolve, or stack parameterized sections.
7	Apply engineering constraints	Enforce minimum thickness, clearance, balance, flow, stress, and manufacturing limits.
8	Optimize and iterate	Use parameter sweeps, simulation, or gradient/evolutionary optimization.
9	Prototype and validate	Fabricate, measure, compare with CAD, and update the model if necessary.

13. Illustrative Case Study: Parametric Radial Air Vent

Consider a circular product vent with six dominant lobes. A conceptual centerline may be defined by $r(\theta)=R_0+A|\cos(6\theta)|$. Let $R_0=25$ mm and $A=18$ mm. The parameter 6 controls the number of repeated lobes, A controls visual depth, and R_0 preserves a central clearance zone. A phase parameter ϕ can be introduced as $r(\theta)=R_0+A|\cos(6\theta+\phi)|$ to rotate the pattern. The curve is then offset by a specified rib width and clipped between inner and outer circular boundaries.

Case Study: Parametric Six-Lobe Radial Air Vent

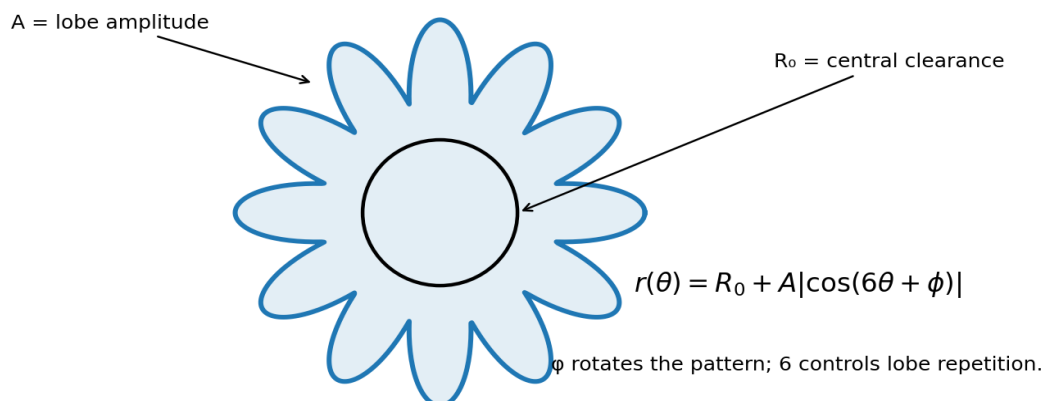


Figure 13. Six-lobe radial air-vent case study showing the geometric roles of R_0 , A, and phase ϕ .

The design is immediately scalable and variant-friendly: changing the lobe count from 6 to 8 creates a new visual identity while preserving the overall design logic. Before production, the designer would calculate open-area ratio, check minimum rib thickness, verify tooling constraints, and run airflow/acoustic simulations where relevant. This case illustrates the central argument of this paper: a polar graph can act as a compact design genome whose parameters drive an entire product family.

14. Discussion

Polar-coordinate design is strongest when the underlying product logic is radial or rotational. It reduces geometric complexity, improves parameter transparency, and makes systematic variation easier. It also creates a natural bridge between mathematics education and professional computational design: equations become editable geometry, and geometry becomes manufacturable data.

Nevertheless, polar coordinates should not be forced onto arbitrary free-form products. Cartesian, cylindrical, surface-based, subdivision, implicit, or topology-optimization representations may be more suitable depending on the problem. The most effective industrial workflow is hybrid: use polar coordinates where they clarify the design intent, then integrate them with splines, NURBS, solids, simulation, and process-specific constraints.

15. Conclusion

Polar coordinate graphs offer industrial designers a powerful method for generating and controlling radial geometry. Circles establish hubs and roots; rose curves generate repeated visual structures; spirals control progressive paths and blade sweep; Fourier-type radial functions describe cams and optimized periodic boundaries. Through $x=r \cos\theta$ and $y=r \sin\theta$, these mathematical models transfer directly into CAD-compatible geometry, where they can be lofted, surfaced, optimized, simulated, and manufactured.

The broader literature on turbine and wind-turbine CAD, design automation, reverse engineering, and CAM confirms that parameterized geometric representations are central to modern industrial workflows. Accordingly, polar graphs should be viewed not merely as classroom plots but as practical generative tools for product architecture, engineering iteration, and design automation.

References

1. A differentiated geometry blade parameterization methodology for gas turbines. *Computers & Fluids*, 2025. DOI: 10.1016/j.compfluid.2025.106588.
2. Innovative approach to computer-aided design of horizontal axis wind turbine blades. *Journal of Computational Design and Engineering*, 2017, 4(2), 98–105. DOI: 10.1016/j.jcde.2016.11.001.
3. A Unified Geometry Parametrization Method for Turbomachinery Blades. *Computer-Aided Design*, 2021, 133, 102987. DOI: 10.1016/j.cad.2020.102987.
4. Parametrically automated 3D design and manufacturing for spiral-type free-form models in an interactive CAD/CAM environment. *CAD/CAM and manufacturing research*.
5. A Case Study in CAD Design Automation. *Journal of Technology Studies*, Virginia Tech.
6. Chen, S.-L., & Wang, W.-T. Computer aided manufacturing technologies for centrifugal compressor impellers. *Journal of Materials Processing Technology*, 2001, 115(3), 284–293. DOI: 10.1016/S0924-0136(01)00828-7.

7. Gagnon, L., & Lutz, T. Data for: Transforming Laser-Scanned 750 kW Turbine Surface Geometry Data into Smooth CAD for CFD Simulations. DaRUS, 2024. DOI: 10.18419/DARUS-3859.

Appendix A. Example Polar Equations for Design Exploration

Design form	Equation
Circular datum	$r = R$
Six-lobe grille	$r = R_0 + A \cos(6\theta) $
Three-petal feature	$r = A \cos(3\theta)$
Archimedean spiral	$r = a + b\theta$
Logarithmic spiral	$r = ae^{(b\theta)}$
Eccentric limaçon	$r = a + b \cos\theta$
General periodic profile	$r = a_0 + \sum (a_n \cos n\theta + b_n \sin n\theta)$